Technical data sheet

FOR THE ROTARY ACTUATOR

HSG 11 SZ

Type:



Art. no.: 3706471 Model Vane-type rotary actuator In principle the actuation has a pressure- and viscosity-dependent internal leak volume flow rate. For example, if an external torque affect the rotary actuator shaft in an idle mode, the rotary actuator shaft shifts slowly from its angle position! HSG: Rotary actuator as basic model without an end position damping and without an internal rotating Series angle limitation. 11 Size Mounting style - Rotary actuator housing single-sided face mounting with thread according to DIN 13-1 - M 24 Strength class of the fastening screws \geq 8.8 involute spline shaft according to DIN 5480-W 170x5,0x32x 8f - Drive shaft end - Centre hole in the drive shaft end DIN 332-2 - D M 30 Connection type pipe thread according to DIN ISO 228-1; A and B: G 1; axial in the rear cylinder cover Installation position arbitrary; Depending on the position of installation and case of application a load may cause running ahead the rotary actuator drive shaft. In such a case, appropriate countermeasures must be taken! Installation instructions see operating instructions Rotary angle limitation An external rotary angle limitation is recommended! The rotary actuator is intended for generation an alternating torque in a stationary application. Intended use 1) max. nominal pressure bar 120 **D** N max min. minimum pressure bar 15 Required for a proper functioning of the load-free drive. **D** M min max. starting pressure without load p_{St max} 7.2 at an output pressure of p = 1 bar bar torque constant 2) specific torque M sp Nm/bar 468.81 theoretical torque M_{th} Nm 56 257 at $\Delta p = p_{N max}$ 2) mechanical efficiency \approx 0.940 at $\Delta p = p_{N \max}$ and $\omega = \omega_{\max}$ 3) n mer at $\Delta p = p_{N max}$ and $\omega = \omega_{max}$ effective torque M_{eff} Nm 52 882 3) number of working chambers z 2 2) nominal angle of rotary grad 292 The internal stop must not be approached! ΦN max. operating angle of rotary grad 290 ΦAmax recom. min. operating angle of rotary 13 If smaler rotating angles are to be realised in continious operation, the grad $\phi_{A min}$ manufacturer must be consulted. $F_{r\,max}$ 40 000 maximum radial force force acting centered on the journal of the drive shaft N $F_{\text{ax}\,\text{max}}$ force acting centrically on the journal of the drive shaft maximum axial force Ν 20 000 ± 10%, incl. oil filling weight ≈ m kg 760.0 mass moment of inertia of drive shaft J_{W0} kgm² 2.36 ± 5%, without other attachments such as hub, coupling, rotation encoder, etc. max. angular speed rad/s 0.9 This corresponds to 52 deg/s or an equivalent rotational speed of n= 9 min⁻¹. ¹) ωmax ²) specific displacement cm³/° 81.82 This results in a theoretical operating volume of V_A = 23 728.6 cm³. Vsp theoretical volume flow rate required Q_{th} l/min 253.3 ²) at ω=ω_{max} max. internal leakage volume flow rate Q L max l/min 0.88 at $\Delta p = p_{N max}$ and $v = 50 \text{ mm}^2/\text{s}$ ³)⁴) effective required volume flow rate Q_{eff} l/min 254.2 at $\Delta p = p_{N max}$, $\omega = \omega_{max}$ and $v = 50 \text{ mm}^2/\text{s}$ ³)⁴) permissible pressure fluid HLP mineral oils according to DIN 51524 T2 -20 – +80 The viscosity range set in operation is to be observed. 1) temperature range of pressure fluid °C θö range of kinematic viscositv mm²/s short-term, the optimum operating viscosity range is $30 - 50 \text{ mm}^2/\text{s}$ 18 - 150 ν cleanliness class of pressure fluid Max. permissible degree of pollution according to ISO 4406 class 18/16/13. To increase service life, we recommend according to ISO 4406 class 17/15/12. °C 0 - +60range of ambient temperature θ, design of component surfaces metallic bright and wetted with anticorrosion agents Subject to technical modifications and error!

¹) The simultaneous occurrence of two or more maximum values of temperature, pressure and angular speed requires the written consent of the manufacturer!

²) Theoretically determined value without manufacturing tolerances and if so an efficiency.

³) Median recorded in test series; an inferential variance is possible.

⁴) In mint condition of the internal seals and their counter-surfaces!

Page 1 of 1 V 2.0 2024-03-01 Technical data sheet Type : HSG 11 SZ Art. no. : 3706471 Hense Systemtechnik GmbH & Co. KG Flottmannstr. 55 D-44807 Bochum

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