Technical data sheet

FOR THE ROTARY ACTUATOR



Type: HSG 11 SP Art. no.: 3806474

Model		Vane-type rotary actuator In principle the actuation has a pressure- and viscosity-dependent internal leak volume flow rate. For example, if an external torque affect the rotary actuator shaft in an idle mode, the rotary actuator shaft shifts slowly from its angle position!			
Series		Rotary actuato angle limitatio	or as basic model without an end position damping and without an internal rotati on.	ing	
Size		11			
Mounting style					
- Rotary actuator housing		single-sided face mounting with thread according to DIN 13-1 - M 24			
- Drive shaft end		Strength class of the fastening screws ≥ 8.8 two parallel keys according to DIN 6885-1 - B40x22x200 (2 x 180°)			
- Centre hole in the drive shaft end		DIN 332-2 - D M 30			
Connection type		pipe thread according to DIN ISO 228-1;			
· ·		A and B: G 1; axial in the rear cylinder cover			
Installation position		arbitrary; Depending on the position of installation and case of application a load may cause running ahead the rotary actuator drive shaft. In such a case, appropriate countermeasures must be taken!			
Installation instructions		see operating instructions			
Rotary angle limitation		An external rotary angle limitation is recommended!			
Intended use		The rota	ry actuator is	intended for generation an alternating torque in a stationary application.	
max. nominal pressure	р _{м тах}	bar	120		1)
min. minimum pressure	p _{Mmin}	bar	15	Required for a proper functioning of the load-free drive.	
max. starting pressure without load	p _{St max}	bar	7.2	at an output pressure of p = 1 bar	
specific torque	M _{sp}	Nm/bai	468.81	torque constant	2)
theoretical torque	M _{th}	Nm	56 257	at Δp=p _{N max}	²)
mechanical efficiency ≈	η _{mec}	-	0.940	at $\Delta p = p_{N \text{ max}}$ and $\omega = \omega_{\text{ max}}$	³)
effective torque	M _{eff}	Nm	52 882	at $\Delta p = p_{N max}$ and $\omega = \omega_{max}$	3)
number of working chambers	Z	_	2		,
nominal angle of rotary	Фи	grad	292	The internal stop must not be approached!	2)
max. operating angle of rotary	Фатах		290	•	,
recom. min. operating angle of rotary	φAmin	grad	13	If smaler rotating angles are to be realised in continious operation, the manufacturer must be consulted.	
maximum radial force	F_{rmax}	N	40 000	force acting centered on the journal of the drive shaft	
maximum axial force	F _{ax max}	N	20 000	force acting centrically on the journal of the drive shaft	
weight ≈	m	kg	760.0	± 10%, incl. oil filling	
mass moment of inertia of drive shaft	J_{W0}	kgm²	2.38	± 5%, without other attachments such as hub, coupling, rotation encoder, etc	
max. angular speed	ω _{max}	rad/s	0.9	This corresponds to 52 deg/s or an equivalent rotational speed of n= 9 min ⁻¹ .	
specific displacement	V _{sp}	cm ³ /°	81.82	This results in a theoretical operating volume of V_A = 23 728.6 cm ³ .	²)
theoretical volume flow rate required	Q _{th}	l/min	253.3	at w=w max	²)
max. internal leakage volume flow rate	-		0.88	at $\Delta p = p_{N max}$ and $v = 50 \text{ mm}^2/\text{s}$	3) ⁴)
effective required volume flow rate	Qeff	l/min	254.2	at $\Delta p = p_{N max}$, $\omega = \omega_{max}$ and $\nu = 50 \text{ mm}^2/\text{s}$	³) ⁴)
permissible pressure fluid	₹ eli	0111111	204.2	HLP mineral oils according to DIN 51524 T2	, ,
temperature range of pressure fluid	9 öı	°C	-20 – +80	The viscosity range set in operation is to be observed.	¹)
- '		mm²/s	18 – 150	short-term, the optimum operating viscosity range is 30 – 50 mm ² /s	,
range of kinematic viscosity cleanliness class of pressure fluid	ν	11111 75	16 - 150	Max. permissible degree of pollution according to ISO 4406 class 18/16/13. To increase service life, we recommend according to ISO 4406 class 17/15/12	! .
range of ambient temperature design of component surfaces	θ	°C	0-+60	metallic bright and wetted with anticorrosion agents	
S P				0	

Subject to technical modifications and error!

¹⁾ The simultaneous occurrence of two or more maximum values of temperature, pressure and angular speed requires the written consent of the manufacturer!

 $^{^{\}rm 2})$ Theoretically determined value without manufacturing tolerances and if so an efficiency.

³⁾ Median recorded in test series; an inferential variance is possible.

⁴⁾ In mint condition of the internal seals and their counter-surfaces!